

## Sheet (4) Solution

### Kinematics of fluid flow (continuity equation)

#### Problem One

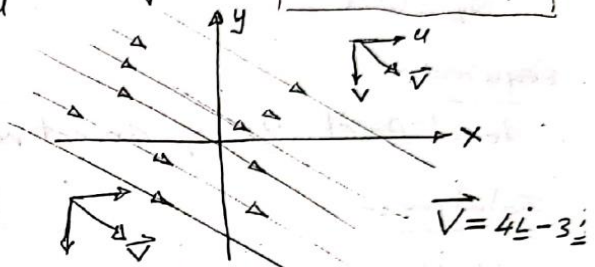
Find the acceleration and stream line function for the flows given below, then sketch the stream lines

(i)  $u=4, v=-3$

$$\left. \begin{aligned} \frac{\partial u}{\partial x} &= 0 & \frac{\partial v}{\partial x} &= 0 \\ \frac{\partial u}{\partial y} &= 0 & \frac{\partial v}{\partial y} &= 0 \end{aligned} \right\} \begin{aligned} \therefore a_x &= 4(0) - 3(0) = 0 \\ \therefore a_y &= 4(0) - 3(0) = 0 \end{aligned}$$

To get stream func.  $\frac{dx}{u} = \frac{dy}{v}$  OR  $\boxed{v dx = u dy}$

$$\begin{aligned} \therefore \int -3 dx &= \int 4 dy \\ \therefore -3x &= 4y + C' \\ \therefore y &= -\frac{3}{4}x + C \quad (\text{line}) \end{aligned}$$

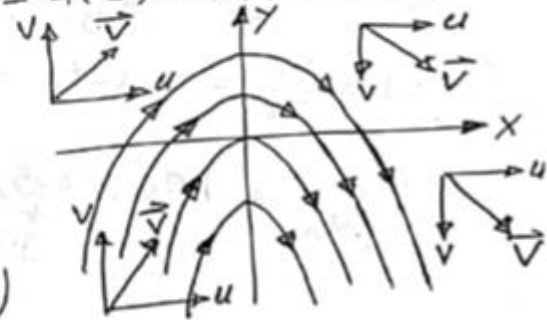


(ii)  $u=4, v=-3x$

$$\left. \begin{aligned} \frac{\partial u}{\partial x} &= 0 & \frac{\partial v}{\partial x} &= -3 \\ \frac{\partial u}{\partial y} &= 0 & \frac{\partial v}{\partial y} &= 0 \end{aligned} \right\} \begin{aligned} \therefore a_x &= 4(0) - 3x(0) = 0 \\ \therefore a_y &= 4(-3) - 3x(0) = -12 \end{aligned}$$

stream func.

$$\begin{aligned} \int -3x dx &= \int 4 dy \\ -\frac{3x^2}{2} &= 4y + C' \\ \therefore y &= -\frac{3}{8}x^2 + C \quad (\text{Parabola}) \end{aligned}$$

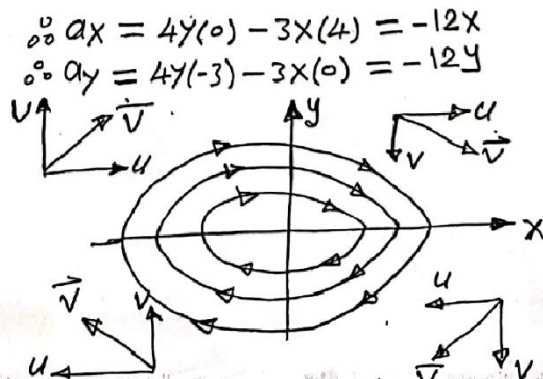


(iii)  $u=4y, v=-3x$

$$\left. \begin{aligned} \frac{\partial u}{\partial x} &= 0 & \frac{\partial v}{\partial x} &= -3 \\ \frac{\partial u}{\partial y} &= 4 & \frac{\partial v}{\partial y} &= 0 \end{aligned} \right\}$$

stream  $\int -3x dx = \int 4y dy$

$$\begin{aligned} \therefore -\frac{3}{2}x^2 &= \frac{4}{2}y^2 + C' \\ \therefore 3x^2 + 4y^2 &= C \quad (\text{Ellipse}) \\ \therefore \frac{x^2}{C/3} + \frac{y^2}{C/4} &= 1 \end{aligned}$$



(iv)  $u=4y, v=3x$

(iv)  $u=4y$  &  $v=3x$

$$\therefore \frac{\partial u}{\partial x} = 0 \quad \frac{\partial v}{\partial x} = 3$$

$$\frac{\partial u}{\partial y} = 4 \quad \frac{\partial v}{\partial y} = 0$$

Stream

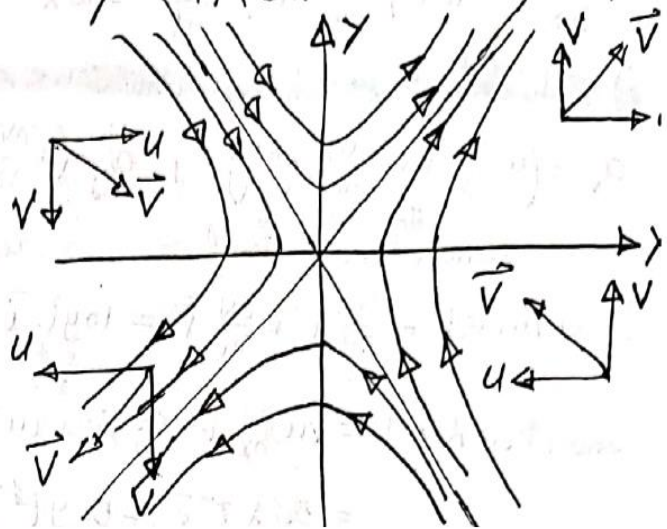
$$\int 3x dx = \int 4y dy$$

$$\therefore \frac{3x^2}{2} = \frac{4y^2}{2} + C'$$

$$\therefore 3x^2 - 4y^2 = C \text{ (Hyperbola)}$$

$$\therefore a_x = 4y(0) + 3x(4) = 12x$$

$$\therefore a_y = 4y(3) + 3x(0) = 12y$$



(v)  $u=4y, v=-4x$

(v)  $u=4y$  &  $v=-4x$

$$\frac{\partial u}{\partial x} = 0 \quad \frac{\partial v}{\partial x} = -4$$

$$\frac{\partial u}{\partial y} = 4 \quad \frac{\partial v}{\partial y} = 0$$

Stream:

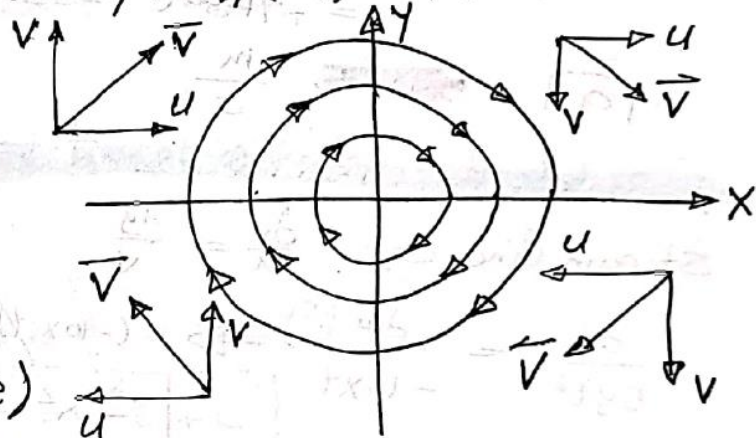
$$\int -4x dx = \int 4y dy$$

$$-\frac{4x^2}{2} = \frac{4y^2}{2} + C$$

$$\therefore x^2 + y^2 = c \text{ (circle)}$$

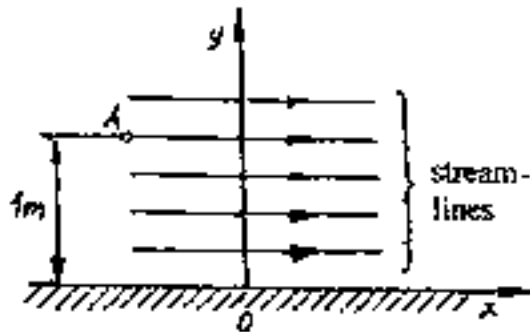
$$\therefore a_x = 4y(0) - 4x(4) = -16x$$

$$\therefore a_y = 4y(-4) - 4x(0) = -16y$$



## Problem two

Unsteady, two dimensional flow,  $V_y=0$ ,  $V_x=5yt^2$ , Calculate the local and convective acceleration in point 'A' at  $t=0.5$  s.



$$V = V_y = 0 \rightarrow a_y = 0$$

$$u = V_x = 5yt^2 \Rightarrow a_x = u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} + \frac{\partial u}{\partial t}$$

$$a_{x, conv} = u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = (5yt^2)(0) + (0)(5t^2) = 0$$

$$a_{local} = \frac{\partial u}{\partial t} = 5y(2t) = 10yt$$

$$\left. \frac{\partial u}{\partial t} \right|_{at A} = 10(1)(0.5) = 5 \text{ m/s}^2$$

From Fig.
Given.

## Problem Three

If the velocity distribution in a circular pipe of radius  $R$  is given by  $u = U_{max} \left(1 - \frac{r^2}{R^2}\right)$  where  $r$  is the radial distance from center and  $U_{max}$  is the maximum flow velocity at the center, find the drag force on a section of pipe 20cm long, 12cm diameter. Use  $U_{max}=3\text{m/s}$  and  $\mu=0.011$  poise. Sketch the velocity distribution.

$$u = U_{max} \left(1 - \frac{r^2}{R^2}\right)$$

drag force = shear force

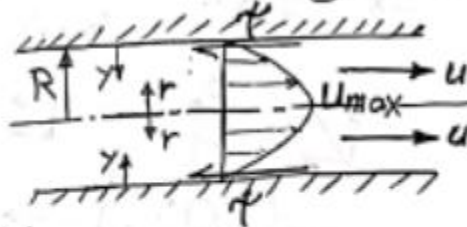
$$\tau = \mu \cdot \frac{du}{dy} = -\mu \cdot \frac{du}{dr} \Big|_{r=R}$$

$$F_{shear} = \tau \cdot A_{surface} (2\pi RL)$$

$$\begin{aligned} \left. \frac{du}{dr} \right|_R &= U_{max} \left(0 - \frac{2r}{R^2}\right) \Big|_R \\ &= U_{max} \frac{-2R}{R^2} = \frac{-2U_{max}}{R} \end{aligned}$$

$$F = -\mu \left( \frac{-2U_{max}}{R} \right) (2\pi RL)$$

$$F_D = 4\pi (0.011 \times 10^{-1}) (3) (20) = 0.829 \text{ N}$$



## Problem four

Unsteady, two dimensional flow,  $u=5yt^2$ ,  $v=-10xt$

- Calculate the local and convective acceleration at point A (-1,1) at  $t=2$  sec
- Calculate the total acceleration at (-1,1) at  $t=2$  sec
- Sketch the stream lines at  $t=2$  sec & 1 sec

$$\begin{array}{l|l} u = 5y t^2 = 5(1)(2)^2 = 20 & v = -10xt = (-10)(-1)(2) = 20 \\ \frac{\partial u}{\partial x} = \text{zero} & \frac{\partial v}{\partial x} = -10t = -20 \\ \frac{\partial u}{\partial y} = 5t^2 = 20 & \frac{\partial v}{\partial y} = \text{zero} \\ \frac{\partial u}{\partial t} = 10yt = 10 \times 1 \times 2 = 20 & \frac{\partial v}{\partial t} = -10x = -10 \times -1 = 10 \end{array}$$

- Calculate the local and convective acceleration at point A (-1,1) at  $t=2$  sec

$$\vec{a} = a_x \vec{i} + a_y \vec{j}$$

$$\begin{array}{l} a_x = \left\{ u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right\} + \left\{ \frac{\partial u}{\partial t} \right\} \\ a_y = \left\{ u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right\} + \left\{ \frac{\partial v}{\partial t} \right\} \\ \text{Convective Accel.} \qquad \qquad \qquad \text{Local Accel.} \end{array}$$

$$\text{Local Accel.} = \frac{\partial u}{\partial t} \vec{i} + \frac{\partial v}{\partial t} \vec{j} = 20 \vec{i} + 10 \vec{j}$$

$$\begin{aligned} \text{Convective Accel.} &= \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) \vec{i} + \left( u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} \right) \vec{j} \\ &= 20 \times 20 \vec{i} + 20 \times (-20) \vec{j} \\ &= 400 \vec{i} - 400 \vec{j} \end{aligned}$$

(ii) Calculate the total acceleration at (-1,1) at t= 2 sec

Total Acceleration = convective Accel. + Local Accel.

$$= 420 \mathbf{i} - 390 \mathbf{j}$$

$$|a| = 573.15 \text{ m/s}^2$$

(iii) Sketch the stream lines at t=2 sec & 1 sec

Stream line eqn  $\frac{dx}{u} = \frac{dy}{v}$

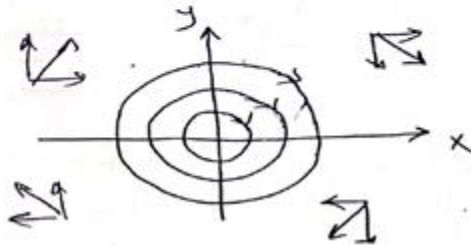
$$\frac{dx}{5yt^2} = \frac{dy}{-10xt} \Rightarrow (-10xt) dx = (5yt^2) dy$$

$$\boxed{-x^2 = \frac{y^2}{2} t + C}$$

at t=2 sec

$$-x^2 = y^2 + C \Rightarrow x^2 + y^2 = C$$

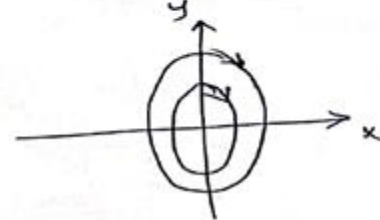
↳ Circle.



at t=1 sec.

$$-x^2 = \frac{y^2}{2} + C$$

$$\frac{x^2}{1} + \frac{y^2}{2} = C$$



## Problem Five

Find and sketch the stream line for the given below and calculate the acceleration at P (-1,2)

(i)  $u=x, v=-y$

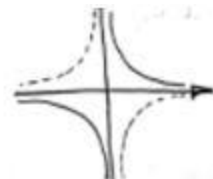
(ii)  $u=x, v=y$

(iii)  $u=y^2, v=xy$

(iv)  $u=y^2, v=-xy$

(v)  $u=x, v=2y$

(i)  $u=x, v=-y$



$$\therefore \int \frac{dx}{x} = \int \frac{dy}{-y}$$


$$\therefore \ln x + \ln y = c'$$

$$\therefore \ln xy = \ln c$$

$$\therefore xy = c$$

$$\therefore y = c/x \text{ (hyperbola)}$$

(ii)  $u=x, v=y$



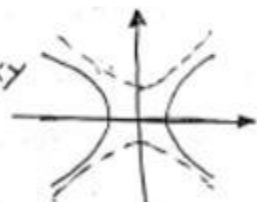
$$\therefore \int \frac{dx}{x} = \int \frac{dy}{y}$$

$$\therefore \ln x + c = \ln y$$

$$\ln cx = \ln y$$

$$\therefore y = cx \text{ (line)}$$

(iii)  $u=y^2, v=xy$




$$\frac{dx}{y^2} = \frac{dy}{xy}$$

$$\int x dx = \int y dy$$

$$\frac{x^2}{2} = \frac{y^2}{2} + c$$

$$x^2 - y^2 = c \text{ (hyperbola)}$$

(iv)  $u=y^2, v=-xy$



$$\frac{dx}{y^2} = \frac{dy}{-xy}$$

$$\int x dx = \int -y dy$$

$$\frac{x^2}{2} + \frac{y^2}{2} = c$$

$$x^2 + y^2 = c \text{ circle}$$

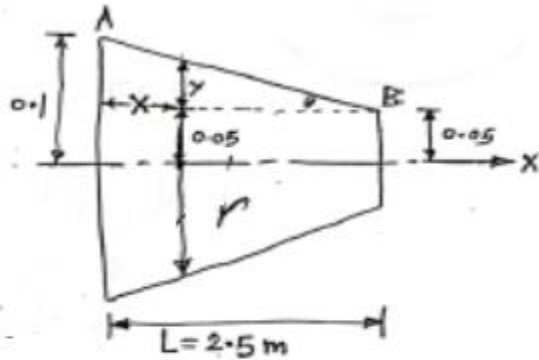
## Problem Six

A conical nozzle 2.5m long converging from 0.2m to 0.1m diameter linearly is subjected to a constant flow rate of  $1 \text{ m}^3/\text{s}$ . Determine the acceleration at the mid-length of the nozzle. Assume uniform flow over each cross section.

$$Q = 1 \text{ m}^3/\text{sec}$$

$$a) \quad \frac{d}{dx} = ?? \quad u) = ??$$

$$\therefore a_x = u \cdot \frac{du}{dx} \quad \text{one dim}$$



$$\text{from } ABC \quad \frac{y}{0.05} = \frac{2.5 - x}{2.5}$$

$$\therefore y = 0.05 - 0.02x$$

$$\therefore r = (y + 0.05) = 0.1 - 0.02x \Rightarrow r = 0.075 \text{ m}$$

$$\therefore A = 0.01767$$

$$\therefore U_x = \frac{Q}{A_x} = \frac{1}{\pi (0.1 - 0.02x)^2}$$

$$\therefore U_x = \frac{1}{\pi} (0.1 - 0.02x)^{-2}$$

$$\therefore \frac{du}{dx} = \frac{-2}{\pi} (0.1 - 0.02x)^{-3} (-0.02)$$

$$U_A = 31.83 \text{ m/s}$$

$$U_B = 127.32 \text{ m/s}$$

$$\text{at } x = 1.25$$

$$\therefore U = \frac{1}{\pi} [0.1 - 0.02(1.25)]^2 = 56.59 \text{ m/s} \quad \#$$

$$\therefore \frac{du}{dx} = \frac{0.04}{\pi} (0.1 - 0.02(1.25))^3 = 30.18 \text{ 1/s}$$

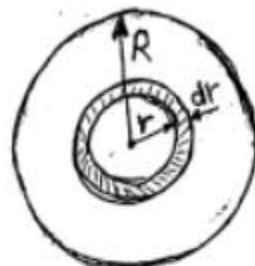
$$\therefore a_x = (56.59)(30.18) = 1707.9 \text{ m/s}^2 \quad \#$$

$$\therefore a_x = u \cdot \frac{du}{dx} = \frac{0.04}{\pi^2 (0.1 - 0.02x)^5}$$

## Problem Seven:

For a laminar flow in a tube where the velocity distribution is given by  $U = U_{max} \left(1 - \frac{r^2}{R^2}\right)$  where  $R$  is the tube radius and  $U_{max}$  is the center line velocity. Show that the mean velocity is half the center line velocity.

$$U = U_{max} \left(1 - \frac{r^2}{R^2}\right)$$
$$\therefore U_{\text{mean avg}} = \frac{\int U \cdot dA}{A}$$
$$\therefore U_{\text{mean avg}} = \frac{\int_0^R U_{max} \left(1 - \frac{r^2}{R^2}\right) \cdot 2\pi r \, dr}{\pi R^2}$$
$$= \frac{2\pi U_{max}}{\pi R^2} \int_0^R \left(r - \frac{r^3}{R^2}\right) dr$$
$$= \frac{2 U_{max}}{R^2} \left(\frac{r^2}{2} - \frac{r^4}{4R^2}\right) \Big|_0^R$$
$$= \frac{2 U_{max}}{R^2} \left(\frac{R^2}{2} - \frac{R^4}{4R^2}\right) = \frac{U_{max}}{2}$$



## Problem Eight:

The velocity vector of a fluid particle is given by

$$u = x^2y + t^2, \quad v = x^2 + y^2 + z^2 + 3t^4, \quad w = x^3 + z^2 + t.$$

Calculate the velocity and acceleration of the particle at point  $(2,3,1)$  after 2 sec

$$P(2,3,1) \quad t = 2 \text{ sec}$$
$$u = x^2y + t^2 \quad v = x^2 + y^2 + z^2 + 3t^4 \quad w = x^3 + z^2 + t$$
$$\frac{\partial u}{\partial x} = 2xy = 12 \quad \frac{\partial v}{\partial x} = 2x = 4 \quad \frac{\partial w}{\partial x} = 3x^2 = 12$$
$$\frac{\partial u}{\partial y} = x^2 = 4 \quad \frac{\partial v}{\partial y} = 2y = 6 \quad \frac{\partial w}{\partial y} = 0$$
$$\frac{\partial u}{\partial z} = 0 \quad \frac{\partial v}{\partial z} = 2z = 2 \quad \frac{\partial w}{\partial z} = 2z = 2$$
$$\frac{\partial u}{\partial t} = 2t = 4 \quad \frac{\partial v}{\partial t} = 12t^3 = 96 \quad \frac{\partial w}{\partial t} = 1$$



To evaluate velocity

$$U = (2)^2(3) + (2)^2 = 16$$

$$V = (2)^2 + (3)^2(1)^2 + 3(2)^4 = 62$$

$$W = (2)^3 + (1)^2 + 2 = 11$$

$$\therefore \bar{V} = 16\hat{i} + 62\hat{j} + 11\hat{k}$$

$$\therefore |\bar{V}| = \sqrt{(16)^2 + (62)^2 + (11)^2} = 64.97 \text{ m/s}$$

$$\therefore a_x = U \frac{\partial U}{\partial x} + V \frac{\partial U}{\partial y} + W \frac{\partial U}{\partial z} + \frac{\partial U}{\partial t} = 444 \text{ m/s}^2$$

$$\therefore a_y = U \frac{\partial V}{\partial x} + V \frac{\partial V}{\partial y} + W \frac{\partial V}{\partial z} + \frac{\partial V}{\partial t} = 554 \text{ m/s}^2$$

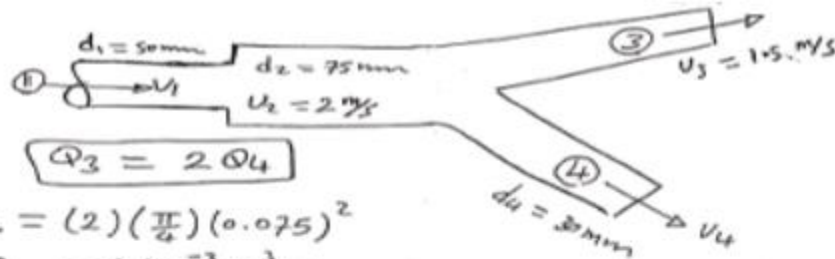
$$\therefore a_z = U \frac{\partial W}{\partial x} + V \frac{\partial W}{\partial y} + W \frac{\partial W}{\partial z} + \frac{\partial W}{\partial t} = 215 \text{ m/s}^2$$

$$\therefore \bar{a} = 444\hat{i} + 554\hat{j} + 215\hat{k}$$

$$\therefore |\bar{a}| = \sqrt{(444)^2 + (554)^2 + (215)^2} = 741.8 \text{ m/s}^2$$

## Problem Nine

Water flows through the shown pipe system. Such that the discharge  $Q_2$  divides so that  $Q_3 = 2Q_4$ . Calculate the values  $Q_1$ ,  $Q_2$ ,  $Q_3$ ,  $Q_4$ ,  $V_1$ ,  $V_4$  and  $d_3$ .



$$Q_1 = Q_2 = (2) \left( \frac{\pi}{4} \right) (0.075)^2$$

$$Q_1 = Q_2 = 8.84 \times 10^{-3} \text{ m}^3/\text{s} \quad \#$$

$$\therefore V_1 = \frac{8.84 \times 10^{-3}}{\frac{\pi}{4} (0.050)^2} = 4.5 \text{ m/s} \quad \#$$

$$\because Q_2 = Q_3 + Q_4$$

$$\because 8.84 \times 10^{-3} = 3Q_4 \quad \rightarrow \quad Q_4 = 2.95 \times 10^{-3} \text{ m}^3/\text{s}$$

$$\text{so } Q_3 = 5.9 \times 10^{-3} \text{ m}^3/\text{s}$$

$$\because Q_3 = \frac{\pi}{4} d_3^2 \cdot V_3$$

$$\because 5.9 \times 10^{-3} = \frac{\pi}{4} (1.5) d_3^2 \quad \rightarrow \quad d_3 = 70.8 \text{ mm}$$

$$\because V_4 = \frac{2.95 \times 10^{-3}}{\frac{\pi}{4} (0.030)^2} \quad \rightarrow \quad V_4 = 4.17 \text{ m/s}$$

## Problem Ten

An incompressible fluid flows steadily through a duct which has two outlets. The flow is one-dimension at section (1) (2), but the velocity profile is Parabolic at section (3). What is the velocity  $V_1$ ?

$V_1 = ??$   
 $A_1 = 0.3 \text{ m}^2$   
 $A_2 = 0.2 \text{ m}^2$   
 $V_2 = 1 \text{ m/s}$   
 $A_3 = 0.1 \text{ m}^2$   
 $V_3 = 4(1 - \frac{r^2}{R^2}) \text{ m/s}$   
 $\bar{V}_3 = \frac{4}{2}$

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المساواة

$$U_3 = \frac{4}{2} = 2 \text{ m/s}$$

$$\rho U_1 A_1 = U_2 A_2 + U_3 A_3$$

$$\rho U_1 (0.3) = (1)(0.2) + (2)(0.1)$$

$$\therefore U_1 = \frac{4}{3} \text{ m/s}$$

## Problem Eleven

An airplane moves forward at a speed of 971 Km/hr. The frontal intake area of the jet engine is  $0.8 \text{ m}^2$  and the entering air density is  $0.736 \text{ Kg/m}^3$ . A stationary observer estimates that relative to the earth, the jet engine exhaust gases moves away from the engine with speed of 1050 Km/hr. The engine exhaust area is  $0.558 \text{ m}^2$  and the exhaust gas density is  $0.515 \text{ Kg/m}^3$ . Estimate the mass flow rate of the fuel into the engine in Kg/hr.

$V_1 = 971 \frac{\text{km}}{\text{hr}}$   
 $A_1 = 0.8 \text{ m}^2$  intake  
 $\rho_1 = 0.736 \frac{\text{kg}}{\text{m}^3}$

$V_2 = 1050 + 971 = 2021 \frac{\text{km}}{\text{hr}}$  exhaust  
 $A_2 = 0.558 \text{ m}^2$   
 $\rho_2 = 0.515 \frac{\text{kg}}{\text{m}^3}$

سرعة انفارات  
والطائرة ساكنة على الارض  
اتجاه الطيران

المساواة

$$\dot{m}_p = ??$$

$$\rho_1 V_1 A_1 + \dot{m}_p = \rho_2 V_2 A_2$$

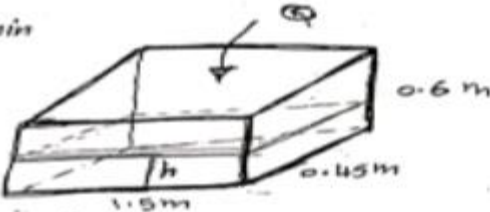
$$(0.736)(971 \times 10^3)(0.8) + \dot{m}_p = (0.515)(2021 \times 10^3)(0.558)$$

$$\therefore \dot{m}_p \approx 9050 \text{ Kg/hr}$$

## Problem Twelve

A bathtub is being filled with water from a faucet. The rate of flow,  $Q$  From the faucet is steady at  $0.0045 \text{ m}^3/\text{min}$ . The tub volume is approximated by a rectangular space. Determine the rate of change of the depth of water in the tub in  $\text{m}/\text{min}$  at any instant

$Q = 0.0045 \text{ m}^3/\text{min}$   
 $\dot{m} = 4.5 \text{ kg}/\text{min}$   
Continuity equ



$$\oiint_S \rho \cdot \mathbf{u} \cdot d\mathbf{A} + \frac{\partial}{\partial t} \iiint_V \rho \cdot dV = 0$$

$\therefore -0.0045 + 0 + \frac{\partial}{\partial t} [h(1.5 \times 0.45)] = 0$

$\therefore \frac{\partial h}{\partial t} = \frac{0.0045}{0.675} = 6.67 \times 10^{-3} \text{ m}/\text{min}$   
 $= 6.67 \text{ mm}/\text{min}$

حساب الزمن اللازم لملئ الحزانة

$\therefore \frac{dh}{dt} = 6.67 \times 10^{-3} \text{ m}/\text{min}$

$\therefore \int_0^{0.6} dh = \int_0^T 6.67 \times 10^{-3} dt$

$\therefore (0.6 - 0) = 6.67 \times 10^{-3} T$

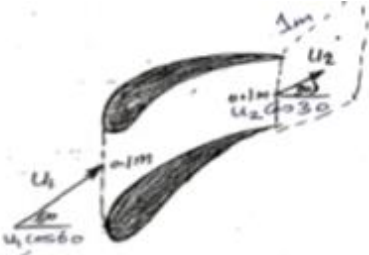
$\therefore T \approx 90 \text{ min}$   
 $= 1.5 \text{ hr}$

## Problem Thirteen

An incompressible fluid flows steadily between a pair of vanes as shown in figure, the average velocity at entrance to the vane is  $10 \text{ m}/\text{s}$ . Determine the volumetric flow rate per unit depth between the vanes and the average velocity at outlet

$u_1 = 10 \text{ m}/\text{s}$   
 $Q = ??$   
 $u_2 = ??$

sol:



$u_1 \cos 60 (A_1) = u_2 \cos 30 (A_2)$

$\therefore u_2 = 10 \cos 60 / \cos 30 = 5.77 \text{ m}/\text{s}$

$\therefore Q = (10 \cos 60)(0.1 \times 1)$   
 $= 0.5 \text{ m}^3/\text{sec}/\text{m wide}$

## Problem Fourteen

Oil ( $S=0.91$ ) enters at section (1) in the shown figure at a weight flow rate of 250N/hr to lubricate the thrust bearing. Oil exits radial through the narrow clearance between thrust plates. Calculate the outlet volume flow rate and the outlet average velocity.

$$m_1 = \frac{250 \cdot \dot{\omega}}{g = 9.81} = m_2$$

$$= 25.48 \text{ Kg/h}$$

$$\therefore Q_2 = \frac{25.48}{910} = \frac{m^3}{s}$$

$$Q_2 = 0.028 \text{ m}^3/\text{hr}$$

$$\therefore Q_2 = V_2 \cdot (\text{Thickness}) \cdot A_2$$

$$\therefore 0.028 = V_2 (\pi (0.10) (0.002))$$

$$\therefore V_2 = 44.56 \text{ m/hr}$$

$$= 1.24 \text{ cm/sec}$$

