

(1) Water and alcohol are mixed in the y-duct shown in figure. What is the average density and average velocity? If the 0.2m diameter pipe tapers to 0.16m diameter and the losses encountered are $\frac{3v^2}{2g}$, where v is the velocity at pipe inlet, find the final pressure

Req:

$\rho_3 = ??$

$V_3 = ??$

$P_4 = ??$

$d_1 = 0.1 \text{ m}$
 $V_1 = 3 \text{ m/s}$
 $\rho_1 = 1000 \text{ kg/m}^3$

$d_2 = 0.1 \text{ m}$
 $V_2 = 2 \text{ m/s}$
 $\rho_2 = 800 \text{ kg/m}^3$

$P_3 = 100 \text{ kPa}$ $d_4 = 0.16 \text{ m}$

$d_3 = 0.2 \text{ m}$

losses = $3 \frac{V_3^2}{2g}$
 3-4

$V = Q$

incompressible

السوائل فقط

Sol: $m_1 + m_2 = m_3$ & $V_1 + V_2 = V_3$

$m_3 = \rho_1 V_1 A_1 + \rho_2 V_2 A_2$ & $V_3 = V_1 A_1 + V_2 A_2$

$m_3 = (10^3)(3) \frac{\pi}{4} (0.1)^2 + (800)(2) \frac{\pi}{4} (0.1)^2 = 36.128 \text{ kg/sec}$

$V_3 = (3) \frac{\pi}{4} (0.1)^2 + (2) \frac{\pi}{4} (0.1)^2 = 0.03927 \text{ m}^3/\text{sec}$

$\rho_3 = \frac{m_3}{V_3} = \frac{36.128}{0.03927} \approx 920 \text{ kg/m}^3$

$V_3 = \frac{V_3}{A_3} = \frac{0.03927}{\frac{\pi}{4} (0.2)^2} = 1.25 \text{ m/s}$

Bernoulli's eqn

between ④ ③

$V_3 A_3 = V_4 A_4 \rightarrow V_4 = 1.25 \left(\frac{0.2}{0.16} \right)^2 = 1.95 \text{ m/s}$

$\frac{V_3^2}{2g} + \frac{P_3}{\rho_3 g} = \frac{V_4^2}{2g} + \frac{P_4}{\rho_3 g} + \frac{3V_3^2}{2g}$ losses

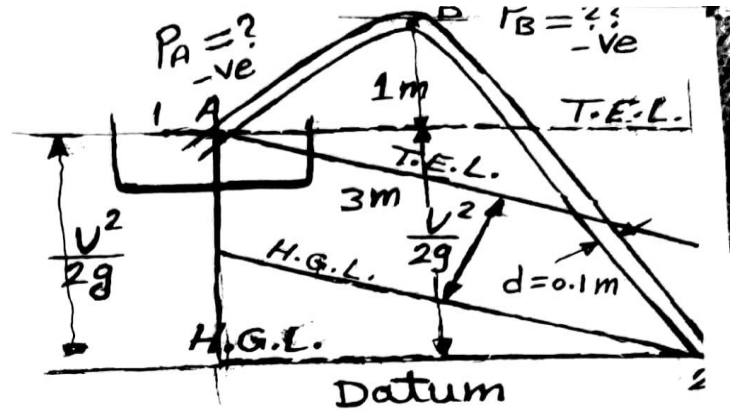
$\frac{(1.25)^2}{2} + \frac{10^5}{920} = \frac{(1.95)^2}{2} + \frac{P_4}{920} + \frac{3(1.25)^2}{2}$

$P_4 = 96813 \text{ Pa}$
 $\approx 96.813 \text{ kPa}$

(2) For the siphon shown in Figure. Find the volume flow rate and the pressure at A, B. Sketch the T.E.L.

[If the loss in the pipe is $\frac{3v^2}{2g}$. Sketch the T.E.L. and H.G.L.]

2] (i) $V = Q = ??$
 $P_A = ??$ $P_B = ??$
 T.E.L. & H.G.L.
 (ii) losses = $\frac{3V^2}{2g}$



Sol: (i) **No losses** Bernoulli (1) (2)

$$\frac{V_1^2}{2g} + \frac{P_1}{\rho g} + z_1 = \frac{V_2^2}{2g} + \frac{P_2}{\rho g} + z_2$$

$\underset{=0}{\frac{V_1^2}{2g}} + \underset{=0}{\frac{P_1}{\rho g}} + \underset{3}{z_1} = \underset{=0}{\frac{V_2^2}{2g}} + \underset{=0}{\frac{P_2}{\rho g}} + \underset{=0}{z_2}$

$\therefore V_2 = \sqrt{2g(3)} = 7.67 \text{ m/s} = V_A = V_B$

$\therefore Q = V = V \cdot \frac{\pi}{4} d^2 = (7.67) \frac{\pi}{4} (0.1)^2 = 0.060 \text{ m}^3/\text{s}$
 $\underline{60 \text{ Litre/sec}}$

Bernoulli (A) (2)

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_A = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_B$$

$\therefore \frac{P_A}{(1000)(9.81)} + 3 = 0 \rightarrow \boxed{\therefore P_A = -29.43 \text{ KPa}}$

Bernoulli (1) (B)

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_1 = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_B$$

$\therefore z = \frac{(7.67)^2}{2g} + \frac{P_B}{(1000)g} + 4 \quad \boxed{\therefore P_B = -39.22 \text{ KPa}}$

(ii) **With losses**

Bernoulli (1) (2)

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_1 = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + z \right)_2 + h_{l_{1-2}}$$

$0 + 0 + 3 = \frac{V^2}{2g} + 0 + 0 + \frac{3V^2}{2g}$

$\therefore Q = V \times \frac{\pi}{4} d^2$

$\therefore V = 3.836 \text{ m/s}$
 $\therefore Q = 0.030 \text{ m}^3/\text{s}$
 $\underline{30 \text{ Litre/sec}}$

Bernoulli (A) (2):

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_A = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_2 + h_{L_{A2}} \frac{3V^2}{2g}$$

$$\therefore \frac{P_A}{(1000)g} + 3 = 0 + \frac{3(3.836)^2}{2g}$$

$$\therefore P_A = -7.358 \text{ KPa}$$

Also:

Bernoulli (A):

$$0 + 0 + 3 = \frac{(3.836)^2}{2g} + \frac{P_A}{(1000)g}$$

$$\therefore P_A = -7.357 \text{ KPa}$$

Bernoulli (A) (B):

$$h_{L_{1-B}} = \frac{1}{3} h_{L_{1-2}} = \frac{1}{3} \left(\frac{3V^2}{2g} \right) = \frac{V^2}{2g}$$

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_1 = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_B + h_{L_{1-B}}$$

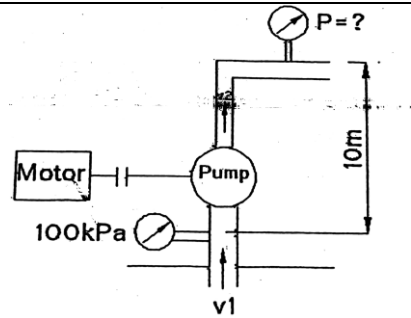
$$\therefore 3 = \frac{(3.836)^2}{2g} + \frac{P_B}{(1000)g} + 4 + \frac{(3.836)^2}{2g}$$

$$\therefore P_B = -9.81 \text{ KPa}$$

لاحظ: زون P_A و P_B تتأثر بالسرعة داخل الأنبوب
كلما زادت السرعة قل الضغط

Problem (3)

The water pump shown in figure is powered by a 15KW motor with an efficiency of 90%. If the flow rate is 0.3 m³/min, inlet pipe is 20cm diameter and outlet pipe is 15cm diameter, find the gage reading, given the pressure at inlet is 100KPa, and the loss is $\frac{3v_2^2}{2g}$



3] Power = Q · ρ · g · h_p

$Q = 0.3/60 = 5 \times 10^{-3} \text{ m}^3/\text{sec}$
 $Q = V_1 A_1 = V_2 A_2$

Power of pump = 15000 × 0.9 = 13500 W

$(1000) \left(\frac{0.3}{60} \right) (9.81) \text{ Hp} = 13500$

∴ Hp = 275.23 m

given: $h_L = \frac{3V_2^2}{2g}$

∴ $V_1 = \frac{Q}{A_1} = \frac{5 \times 10^{-3}}{\frac{\pi}{4} (0.2)^2} = 0.159 \text{ m/sec}$

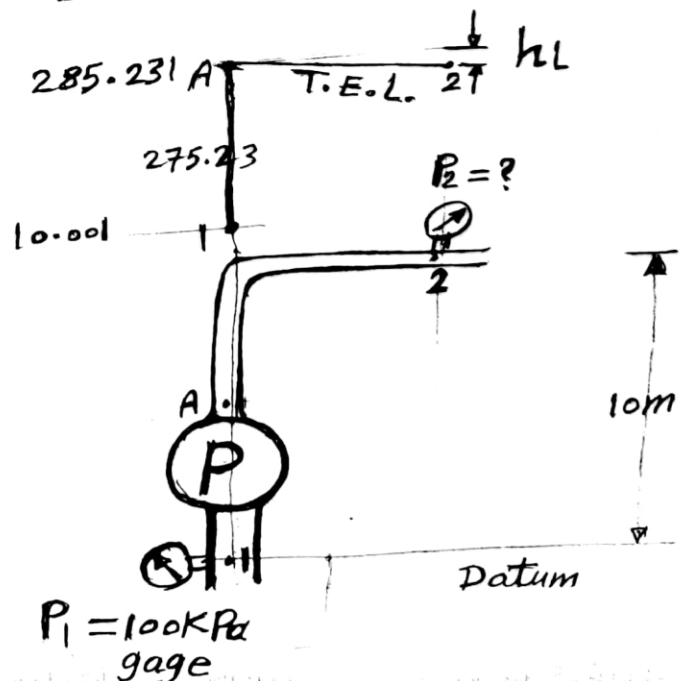
∴ $V_2 = \frac{5 \times 10^{-3}}{\frac{\pi}{4} (0.15)^2} = 0.283 \text{ m/sec}$

Bernoulli ① ② ∴ P₂ = ??

$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_1 + H_p = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_2 + h_L$

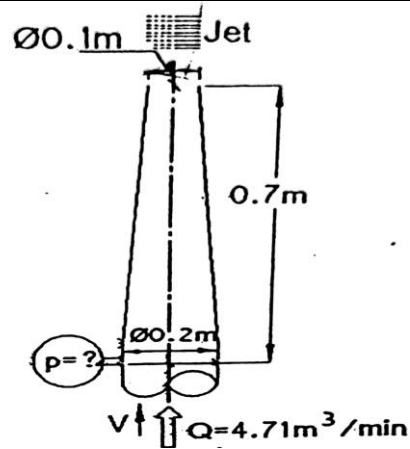
$\frac{(0.159)^2}{2g} + \frac{10^5}{10^3 g} + 0 + 275.23 = \frac{(0.283)^2}{2g} + \frac{P_2}{10^3 g} + 10 + \frac{3(0.28)^2}{2g}$

∴ P₂ = 2701.8 KPa ≈ 2.7 MPa



Problem (4)

Find The gage reading, p , shown in figure, when water is flowing at the rate of $4.71 \text{ m}^3/\text{min}$, if the losses in the nozzle are $\frac{v^2}{4g}$ and v is the inlet velocity. Find the maximum height of the jet.



[4] $Q = 4.71/60 = 0.0785 \text{ m}^3/\text{sec}$

$\therefore V_1 = \frac{0.0785}{\frac{\pi}{4}(0.2)^2} = 2.50 \text{ m/s}$

$\therefore V_2 = \frac{0.0785}{\frac{\pi}{4}(0.1)^2} = 10 \text{ m/s}$

Bernoulli ①②:

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z\right)_1 = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z\right)_2 + h_{L_{1-2}}$$

$\therefore \frac{(2.5)^2}{2g} + \frac{P_1}{10^3 \times g} + 0 = \frac{(10)^2}{2g} + 0 + 0.7 + \frac{(2.5)^2}{4g}$

$\therefore P_1 = 55.30 \text{ kPa}$

Bernoulli ②③:

$$\frac{(10)^2}{2g} + 0 + 0 = 0 + 0 + Z$$

$\therefore Z = 5.097 \text{ m}$

Bernoulli ②④: $\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z\right)_2 = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z\right)_4$

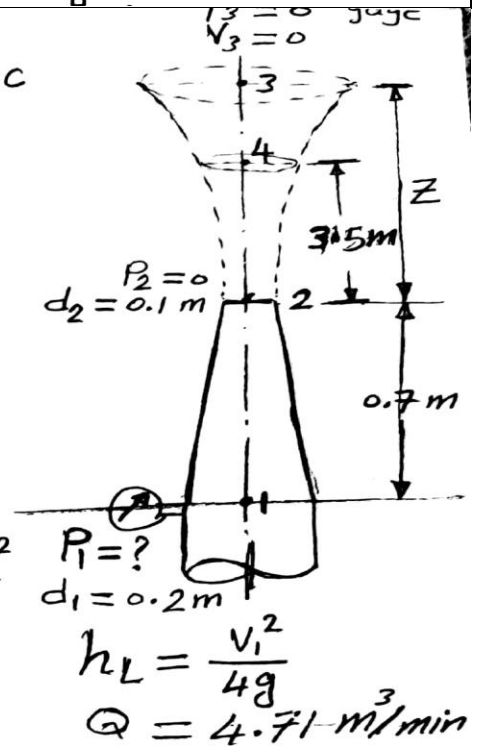
$\therefore \frac{(10)^2}{2g} + 0 + 0 = \frac{V_4^2}{2g} + 0 + 3.5$

$\therefore V_4 = 5.6 \text{ m/s} \quad \& \quad Q = 0.0785$

$\therefore Q = V_4 \cdot \frac{\pi}{4} d_4^2$

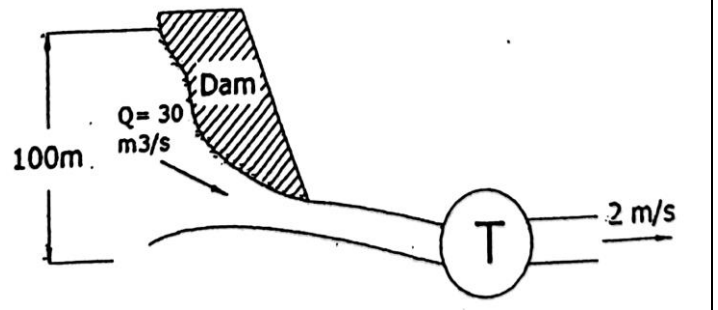
$\therefore 0.0785 = (5.6) \frac{\pi}{4} d_4^2$

$\therefore d_4 = 0.134 \text{ m}$



Problem (5)

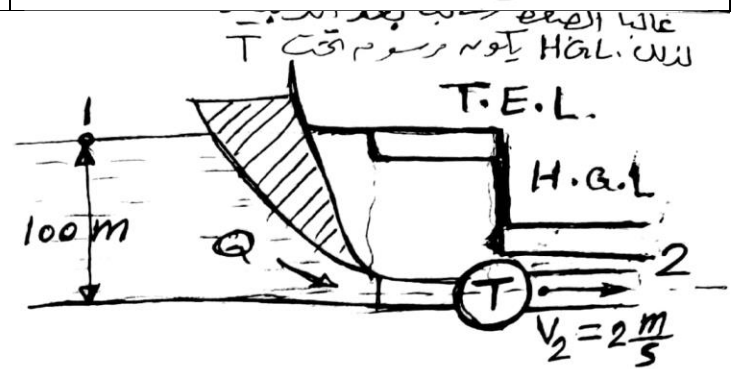
A hydraulic power plant operates under the conditions shown in figure. Find the output power of the turbine, if the turbine efficiency is 95%.



5/ $Q = 30 \text{ m}^3/\text{sec}$

Power of turbine = ??

$\eta_T = 95\%$



Sol: Power of Turbine = $\rho \cdot g \cdot H_T \cdot Q \cdot \eta_T$

Bernoulli ①②:

$$\left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_1 - H_T = \left(\frac{V^2}{2g} + \frac{P}{\rho g} + Z \right)_2 + h_{L_{1-2}}$$

$$\therefore 0 + 0 + 100 - H_T = \frac{(2)^2}{2g} + 0 + 0 + 0$$

$$\therefore H_T = 99.8 \text{ m}$$

$$\therefore \text{Power} = (10^3) (9.81) (99.8) (30) (0.95)$$

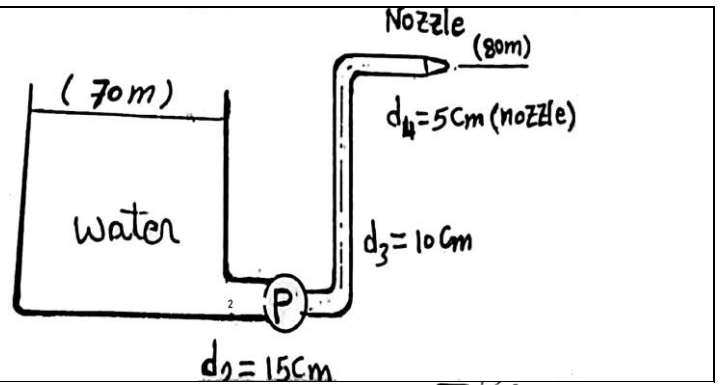
$\text{Power} = 27.9 \text{ MW}$

Notice:

$$\begin{aligned} \text{Fluid Power} &= \rho \cdot g \cdot Q \cdot H_T \\ &= (9810) (30) (99.8) \\ &= 29.37 \text{ MW} \end{aligned}$$

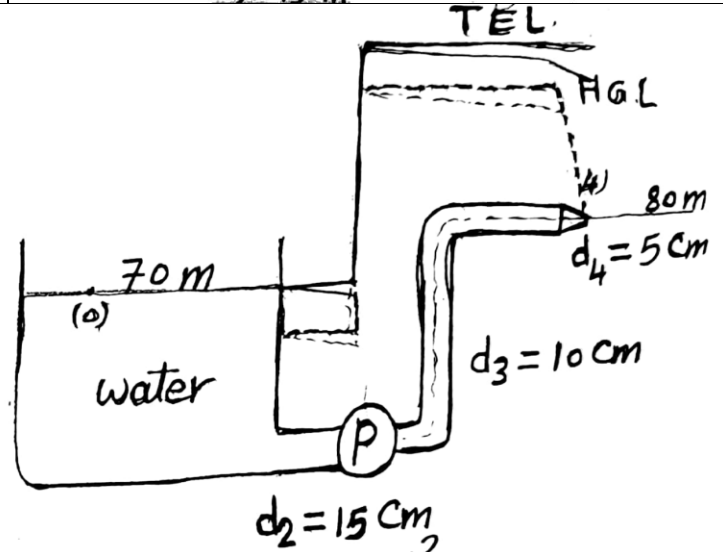
Problem (6)

A pipe line with a pump leads to a nozzle as shown. Find the flow rate when the pump head is 40m assuming that the head losses are $h_2 = 5v_2^2/2g$, $h_3 = 12v_3^2/2g$, $h_4 = 2v_4^2/2g$ then sketch the T.E.L and H.G.L.



6 $H_p = 40 \text{ m}$

$\circ \circ Q = V_2 A_2 = V_3 A_3 = V_4 A_4$
 $\circ \circ V_2 = \frac{1}{9} V_4 \quad \circ \circ V_3 = \frac{1}{4} V_4$
 $\circ \circ h_2 = 0.062 \frac{V_4^2}{2g}$
 $\circ \circ h_3 = 0.75 \frac{V_4^2}{2g}$
 $\circ \circ h_4 = 2 \frac{V_4^2}{2g}$



$\circ \circ h_{\text{loss}} = 2.812 \frac{V_4^2}{2g}$

Bernoulli (0)(4) :

$$\frac{P_0}{\gamma} + \frac{V_0^2}{2g} + Z_0 + H_p = \frac{P_4}{\gamma} + \frac{V_4^2}{2g} + Z_4 + h_{\text{losses}}$$

$$0 + 0 + 70 + 40 = 0 + \frac{V_4^2}{2g} + 80 + 2.812 \frac{V_4^2}{2g}$$

$\circ \circ V_4 = 12.426 \text{ m/s}$

$\circ \circ Q = V_4 A_4 = 0.0244 \text{ m}^3/\text{s}$
 $Q = 24.4 \text{ Litres/s}$

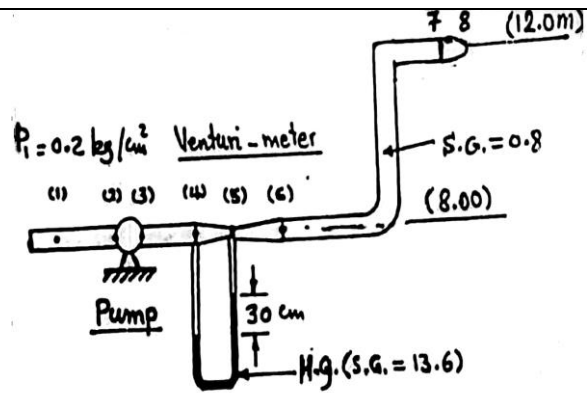
Problem (7)

For the shown pump, venture-meter and nozzle system, if $d_1 = d_2 = d_3 = d_4 = d_6 = d_7 = 20\text{cm}$, $d_5 = d_8 = 10\text{cm}$, $p_1 = 0.2\text{ bar}$,

Find: (a) The discharge in (m^3/sec).

(b) The pump power in (kW).

(c) Sketch the T.E.L. and the H.G.L.



Venturimeter:

$$Q = \frac{A_4 A_5}{\sqrt{A_4^2 - A_5^2}} \sqrt{2g \left(\frac{\rho_m}{\rho} - 1 \right) h_m}$$

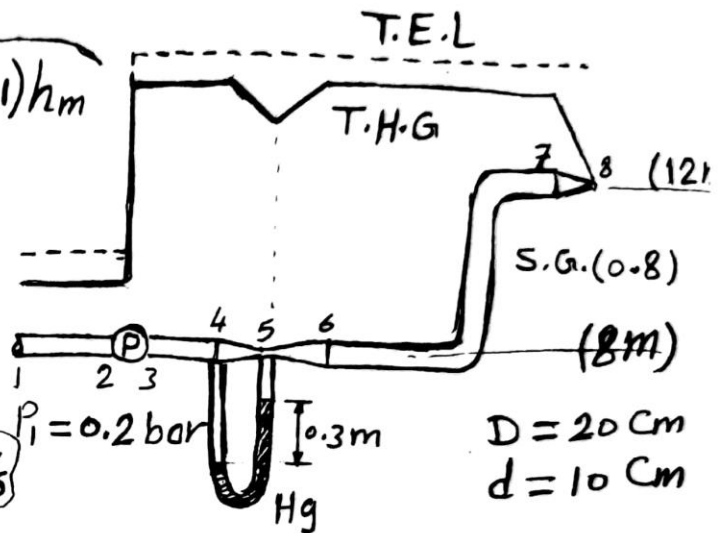
$$A_1 = A_4 = \frac{\pi}{4} (0.2)^2 = 0.03142 \text{ m}^2$$

$$A_8 = A_5 = \frac{\pi}{4} (0.1)^2 = 0.00785 \text{ m}^2$$

$$Q = 0.0787 \text{ m}^3/\text{s}$$

$$V_8 = V_5 = \frac{0.0787}{0.00785} = 10 \text{ m/s}$$

$$V_1 = 2.5 \text{ m/s}$$



Bernoulli (1) (8):

$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 + H_p = \frac{P_8}{\rho} + \frac{V_8^2}{2g} + Z_8$$

$$\frac{0.2 \times 10^5}{(9.81)(800)} + \frac{(2.5)^2}{2(9.81)} + 8 + H_p = 0 + \frac{(10)^2}{2(9.81)} + 12$$

$$H_p = 6.23 \text{ m} \quad \#$$

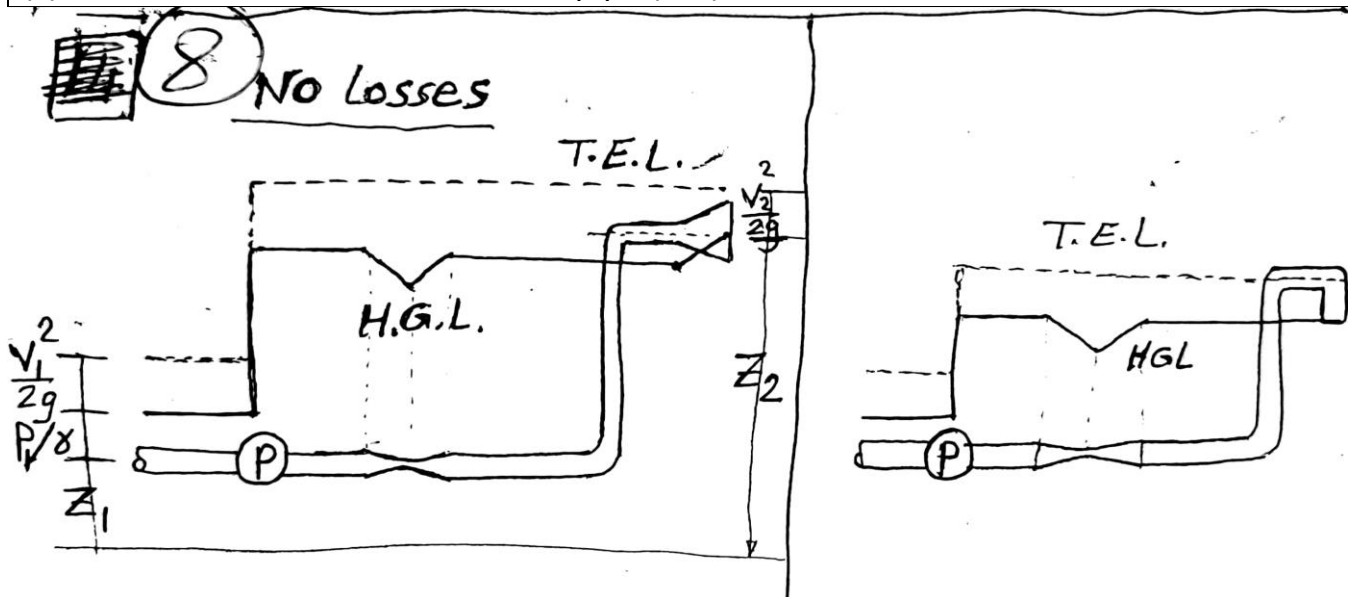
$$\begin{aligned} \text{Power} &= \rho \cdot g \cdot Q \cdot H_p \\ &= (800)(9.81)(6.23)(0.0787) \end{aligned}$$

$$\text{Power} = 3.848 \text{ kW} \quad \#$$

Problem (8)

Sketch only the T.E.L. and the H.G.L. for the previous problem if we:

- (a) Remove the nozzle.
- (b) Replace the nozzle by diffuser.
- (c) Remove the nozzle and bend the pipe (7-8) 90° downward.



Problem (9)

If the water flows steadily through the system shown. Determine the jet velocity, the flow rate and manometric reading h_1 , h_2 and h_3 .

أرشد قوس HGL من أجل لينة الماء

$d_1 = 100 \text{ mm}$
 $d_2 = 50 \text{ mm}$
 $d_3 = 20 \text{ mm (Nozzle)}$

Req: $V_3 = ?$ $Q = ??$
 $h_1 = ?$ $h_2 = ?$ $h_3 = ?$

Sol: $Q = V_1 A_1 = V_2 A_2 = V_3 A_3$

$V_2 = 0.16 V_3$

$V_1 = 0.04 V_3$

Bernoulli (0) (3)

$$0 + 10 + 0 = 0 + 1 + \frac{V_3^2}{2g}$$

$$Q = V_3 \cdot \frac{\pi}{4} d_3^2$$

$$Q = 4.18 \times 10^{-3} \frac{\text{m}^3}{\text{s}}$$

$$\begin{aligned}
 V_3 &= 13.29 \frac{\text{m}}{\text{s}} \\
 V_2 &= 2.13 \frac{\text{m}}{\text{s}} \\
 V_1 &= 0.53 \frac{\text{m}}{\text{s}}
 \end{aligned}$$

Bernoulli (0) (1)

$$0 + 10 + 0 = \frac{P_1}{\gamma} + 15 + \frac{(0.53)^2}{2g}$$

$P_1 = -49.19 \text{ kPa}$

$h_1 = 0.33 \text{ m}$

Manometer:

$$0 = -49.19 \times 10^3 + (1000)(9.81)(0.5) + (13600)(9.81)h_1$$

Bernoulli (0) (2)

$$0 + 10 + 0 = \frac{P_2}{\gamma} + 1 + \frac{(2.13)^2}{2(9.81)}$$

$P_2 = 86 \text{ kPa}$

$h_2 = 0.775 \text{ m}$

Manometer:

$$(13600)(9.81)h_2 = 86 \times 10^3 + (1000)(9.81)(1) + (1000)(9.81)h_2$$

Bernoulli (0) (4) داخل الطاير

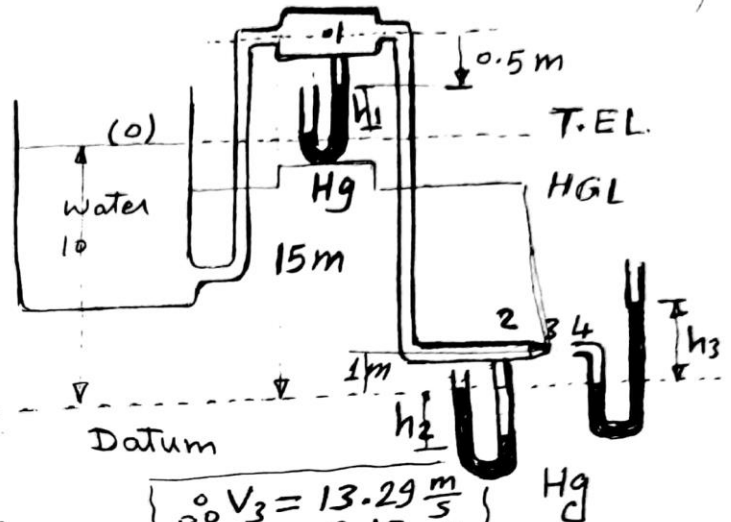
$$0 + 10 + 0 = \frac{P_4}{\gamma} + 1 + 0$$

$P_4 = 88.29 \text{ kPa}$

Manometer:

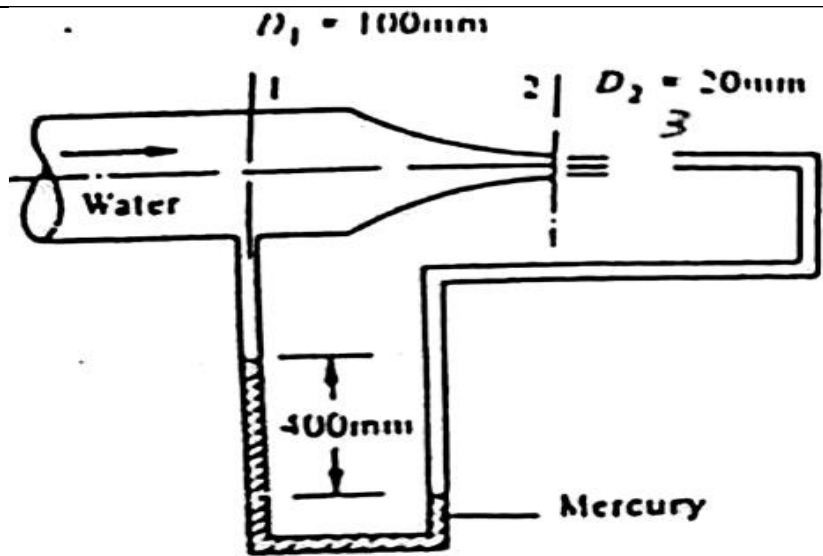
$h_3 = 0.735 \text{ m}$

$$88.29 \times 10^3 + (1000)(9.81)(1) = (13600)(9.81)h_3$$



Problem (10)

Calculate the discharge, Q through the nozzle shown in figure.



10 $D_1 = 100 \text{ mm}$

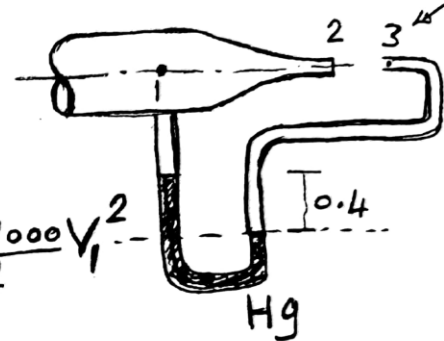
$d_2 = 20 \text{ mm}$

داخل اطار حرة

Bernoulli ① ③ :

$$\frac{P_1}{\gamma} + \frac{V_1^2}{2g} + z_1 = \frac{P_3}{\gamma} + \frac{V_3^2}{2g} + z_3$$

$$\therefore \frac{P_1}{\gamma} + \frac{V_1^2}{2g} = \frac{P_3}{\gamma} \quad \because P_3 = P_1 + \frac{1000}{2} V_1^2$$



Manometer :

$$P_1 + (0.4)(13600)(9.81) = P_3 + (0.4)(1000)(9.81)$$

$$P_1 + 53366.4 = P_3 + \frac{1000}{2} V_1^2 + 3924$$

$\therefore V_1 = 9.94 \text{ m/s}$ //

$\therefore Q = V_1 \cdot A_1$

$\therefore Q = (9.94) \frac{\pi}{4} (0.1)^2$

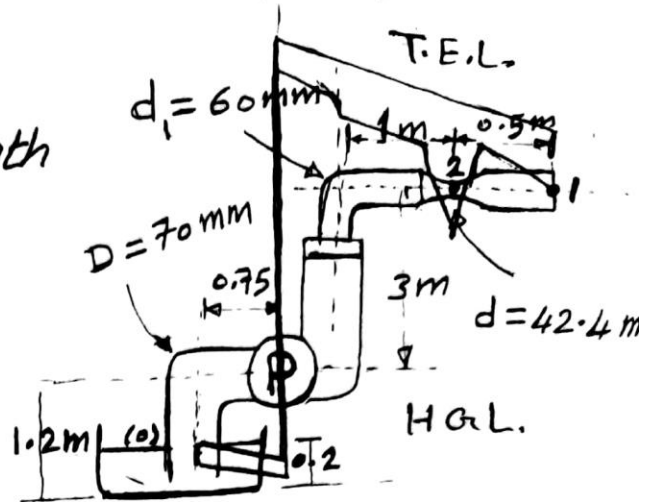
$\therefore Q = 0.0781 \text{ m}^3/\text{sec}$

Problem (11)

In the system shown, the discharge is $0.018 \text{ m}^3/\text{sec}$, energy loss due to friction in the suction and delivery pipes are 0.05 m of water per meter length of the pipe. If the pump efficiency is 0.75 , determine the power required to drive the pump and sketch the T.E.L. and the H.G.L.

4

18 $Q = 0.018 \text{ m}^3/\text{s}$
 $h_L = 0.05 \text{ m water / m length}$
 $\eta_p = 0.75$
 Power = ??



Sol:

$$h_L = (0.05)[1.2 + 0.75 + 3 + 1 + 0.5]$$

$$= 0.323 \text{ m water}$$

$$\therefore V_1 = \frac{Q}{\frac{\pi}{4} d_1^2} = \frac{0.018}{\frac{\pi}{4} (0.06)^2} = 6.37 \text{ m/s}$$

Bernoulli (0)(1)

$$0 + 0 + 0 + H_p = 0 + 4 + \frac{(6.37)^2}{2(9.81)} + 0.323$$

$$\therefore H_p = 6.39 \text{ m water}$$

$$\therefore \text{Power} = \frac{\rho \cdot g \cdot Q \cdot H_p}{\eta_p}$$

$$= \frac{(1000)(9.81)(0.018)(6.39)}{0.75}$$

عزيم

$$= 1.504 \text{ KW} \quad \#$$

Problem (12)

Drive an expression for the flow rate through Venturimeter for steady incompressible flow.

Venturimeter:

① Continuity:

$$V_1 A_1 = V_2 A_2$$

$$V_1 = V_2 \frac{A_2}{A_1} \quad \text{--- ①}$$

② Bernoulli ①②:

$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + z_2$$

$$\frac{P_1 - P_2}{\rho} + (z_1 - z_2) = \frac{V_2^2 - V_1^2}{2g} = \frac{\rho_m}{2g} \left[V_2^2 - V_2^2 \left(\frac{A_2}{A_1} \right)^2 \right]$$

$$\frac{P_1 - P_2}{\rho} + (z_1 - z_2) = \frac{V_2^2}{2g} \left[1 - \left(\frac{A_2}{A_1} \right)^2 \right]$$

$$\therefore V_2 = \frac{A_1}{\sqrt{A_1^2 - A_2^2}} \sqrt{2g \left[\frac{P_1 - P_2}{\rho} + (z_1 - z_2) \right]}$$

$$\therefore Q = V_2 A_2 \quad \therefore Q = \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \sqrt{2g \left[\frac{P_1 - P_2}{\rho} + (z_1 - z_2) \right]}$$

③ Manometer:

$$\therefore P_1 + \rho g z_1 = P_2 + \rho g (z_2 - h_m) + \rho_m \cdot g \cdot h_m$$

$$\therefore (P_1 - P_2) - \rho g (z_2 - z_1) = \rho_m \cdot g h_m - \rho g h_m$$

$$\therefore \frac{P_1 - P_2}{\rho} - (z_2 - z_1) = h_m \left(\frac{\rho_m}{\rho} - 1 \right)$$

$$\therefore Q = \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \sqrt{2g h_m \left(\frac{\rho_m}{\rho} - 1 \right)}$$

$$\begin{aligned} r_1 &> r_2 \\ V_1 &< V_2 \end{aligned}$$

